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Effect of Vaporization on Cryogenic Spray Dropsize Measurement

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EFFECT OF VAPORIZATION ON CRYOGENIC SPRAY DROPSIZE MEASUREMENT

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Abstract

The fluid mechanics of multiphase flow breakup of liquid nitrogen, LN2, jets injected into sonic velocity nitrogen gasflow, was experimentally investigated. A scattered-light scanning instrument was used to measure the characteristic dropsize, D_{v.5}, of LN₂ sprays and to determine the effect of droplet vaporization on experimental dropsize measurements. Under sonic gas-velocity conditions, liquid-jet breakup occurred in the regime of aerodynamic stripping. As a result, the following correlation of volume-median drop diameter, D_{v.5}, with atomizing gas flowrate, Wg, was derived for two-fluid atomizers; $D_{v.5}^{-1} = k_c W_g^n$, where proportionality constant k, and exponent n are functions of droplet vaporization rate. In this study, partially vaporized sprays were investigated and it was found that n = 1.11, which is considerably less than the value of 1.33 that is predicted by atomization theory. This was attributed to the evaporative loss of very small droplets. As a result, the following expression was obtained experimentally: $D_{v.5}^{-1} = 301 W_g^{1.11}$. Values of $D_{v.5}$, that existed prior to partial vaporization of the LN2 sprays, were calculated and the following expression was derived for originally unvaporized LN₂ sprays: $D_{v.5}^{-1} = 285 W_g^{1.33}$. This expression agrees well with atomization theory that predicts n = 1.33, for liquid jet breakup in high-velocity gasflow.

Nomenclature

- a acceleration, cm/sec²
- Cd drag coefficient
- D_{v.5} volume median drop diameter
- k_c correlation coefficient, sec/g-cm
- m mass of characteristic LN₂ droplet, g
- n exponent for weight flow of fluid
- Nu heat transfer Nusselt number, based on D_{v.5}
- Re Reynolds number based on D_{v.5}
- t vaporization time, sec
- V fluid velocity, cm/sec

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- W weight flow of fluid, g/sec
- x axial downstream spray sampling distance
- γ kinematic viscosity, cm²/sec
- ρ fluid density, g/cm³

Subscripts

- a acoustic
- c calculated
- e experimental
- g gaseous nitrogen, GN₂
- l liquid nitrogen, LN2

Introduction

Two-phase flows of liquid nitrogen, LN2, and gaseous nitrogen, GN2, were experimentally investigated. LN2 jets were injected into high-velocity GN2 streams inside a two-fluid fuel nozzle. The jets rapidly disintegrated into clouds of very small droplets. As a result, cryogenic sprays having relatively large droplet surface-areas per unit volume of liquid were produced. High surface-area fuel sprays are desirable since they provide rapid vaporization and efficient combustion of liquid fuel sprays in gas-turbine and rocket combustors. However, in order to calculate vaporization and burning rates of fuel sprays, dropsize data are needed that will accurately represent sprays in terms of a characteristic dropsize, such as the volume-median drop diameter, D_{v.5}, used in this study. Dropsize data were obtained for cryogenic sprays and used to derive correlating expressions for two-fluid atomization, in terms of interacting aerodynamic and liquid surface forces. Such information is needed for analytical modeling of fuelspray combustion processes. Also, fuel-spray dropsize correlations are needed that will cover wide ranges of properties of liquid fuels and atomizing gases. This will aid in the development of more efficient fuel nozzles and thereby improve combustor performance.

Highly volatile cryogenic liquid sprays containing relatively large numbers of very small droplets vaporize quite rapidly. This is due to the presence of high thermal gradients that exist in the gas-film surrounding the droplets. In the present study, the atomizing-gas was close to room temperature, 293 K. As a result, the

original characteristic dropsize initially formed at the atomizer orifice was not measured directly. However, it was possible to determine the original value of $D_{v.5}$, by calculating the amount it changed due to evaporation. Vaporization rates were calculated from heat-transfer and drag coefficient data that had been previously obtained for drops accelerating and vaporizing in high-velocity gasflows. Such data are difficult to obtain, especially when the droplets are microscopic in size and attain relatively high velocity in a very short distance.

In the present study of cryogenic sprays, characteristic dropsize measurements were obtained with a scattered-light scanner. It was originally developed at NASA Lewis Research Center. The instrument was improved optically, as described in Ref. 2. As a result, accurate dropsize measurements were obtained even in the presence of high velocity and high gas-density gradients. Therefore it was possible to determine the effect of droplet vaporization on experimental dropsize measurements, with the improved instrument. Also, it should be noted that sonic gas-velocity conditions were used in this study and liquid-jet breakup occurred in the regime of aerodynamic stripping.

The effect of atomizing gas flowrate on liquid jet breakup in gas streams has been reported in Refs. 1 to 7. In Ref. 1, water sprays were formed with a two fluid atomizer. The volume median drop diameter, $D_{v.5}$, was measured for each spray and correlated with atomizing gas flowrate to give the following general expression: $D_{v.5}^{-1} = k_c W_g^n$, where W_g is GN₂ flowrate. Atomization theory, Ref. 8, predicts that n = 1.33 for the case of liquid jet breakup in high velocity gasflow, i.e., in the regime of aerodynamic stripping.

Numerous investigators, as indicated in Refs. 1 to 7, have studied the effect of gas velocity, $V_{\rm g}$, on liquid-jet breakup in gas streams. Their results are summarized in Table 1. In the study of water-jet breakup that is described in Ref. 1, good agreement was obtained with atomization theory when the exponent n was determined. However, in the case of LN₂ sprays, good agreement with atomization theory may not occur. This can be attributed to the fact that highly volatile sprays can become partially vaporized. In Ref. 1, the study of water sprays showed a marked effect of droplet vaporization on measurements of $D_{\rm v.5}$, when sampling distance downstream of the atomizer orifice was increased from 2.2 to 4.4 and 6.7 cm.

In the present study, original values of $D_{v.5}$ for LN_2 sprays, i.e., prior to their partial vaporization, were calculated over a range of atomizing-gas flowrates. To do this, heat transfer and drag coefficient expressions

obtained from Ref. 9 were used to calculate changes in dropsize that occurred due to partial vaporization of the sprays. Thus, values of $\Delta D_{v.5}$ produced by droplet vaporization could be calculated and used to determine the original value of $D_{v.5}$ that existed before vaporization occurred. LN₂ sprays were injected into a low velocity, 5 m/sec airflow. Fluid pressures for LN₂ and GN₂ were varied over a range of 0.2 to 1.0 MPa and the entire spray was sampled with a 4.4- by 1.9-cm rectangular laser beam. The center line of the laser beam was located at an axial distance of 1.2 cm downstream of the fuel nozzle orifice.

Apparatus and Procedure

As shown in Fig. 1, a two-fluid fuel nozzle was mounted at the center line of the 24-cm diameter duct and operated over pressure ranges of 0.2 to 1.0 MPa for both liquid nitrogen, LN₂, and the atomizing gas, gaseous nitrogen, GN2. LN2 sprays were injected downstream into the airflow. The two-fluid nozzle was fabricated as shown in Fig. 2, with an orifice diameter of 0.66 cm. LN₂ at a temperature of 77 K was axially injected into the airstream by gradually opening the control valve until the desired flowrate of 51 g/s was obtained as indicated by a turbine flowmeter. GN₂ was then turned on and weight flowrate was measured with a 0.51-cm diameter sharp-edge orifice. After the air; atomizing-gas and LN₂ flowrates were set, the volume median diameter of the spray, D_{v.5}, was measured at a distance of 1.5 cm downstream of the atomizer orifice.

The scattered-light scanner consisted of a laser beam expander with spatial filter, rotating scanning slit and detector, as shown in Fig. 1. The instrument measured light as a function of light scatter angle, by repeatedly sweeping a variable-length slit in the focal plane of the collecting lens. The data obtained is scattered-light energy, which is a function of the scatter angle relative to the laser beam axis. This particle size measuring method is similar to that described in Ref. 10.

Measurements of scattered-light energy normalized by the maximum energy and plotted against scattering angle can be used to determine the volume median diameter, $D_{v.5}$, as discussed in Ref. 11 and illustrated in Fig. 3. Dropsize distribution dispersion can also be determined from this plot. Also, according to this method, measurements are independent of particle distribution function. For a typical measurement, the scan is repeated 60 times per second to average out temporal variations in the energy curve. Thus, spray pattern effects were minimized by measuring an average characteristic drop diameter, $D_{v.5}$, for the entire cloud of droplets.

Calibration was accomplished with five sets of monosized polystyrene spheres having diameters of 8, 12, 25, 50, and 100 μ m. LN₂ sprays were sampled fairly close to the atomizer orifice and they contained relatively high number-density droplets. As a result, it was necessary to correct for multiple scattering as described in Ref. 12. Measurement corrections also included Mie scattering theory when very small droplet diameters, i.e., <10 μ m, were measured. Dropsize measurements agreed within ± 5 percent, as indicated by reproducibility tests. Corrections for background scattering caused by high velocity gradients in the laser beam were made by obtaining new background readings, whenever gas velocity was changed and prior to turning on the spray.

Experimental Results

 LN_2 spray measurements of experimental dropsize $D_{v.5e}$ were obtained with the scattered-light scanner located 1.5 cm axially downstream of the two-fluid fuel nozzle. Drop velocity, V_d , was based on $D_{v.5e}$ and drag coefficients were calculated from expressions derived in previous studies using a high-speed droplet tracking camera developed at NACA Lewis laboratory. The technique is described in Ref. 9. Resulting values of V_d and C_d were used to calculate vaporization time Δt , which was then used to determine $\Delta D_{v.5}^2$. Finally, calculated values of $D_{v.5c}$ were obtained and used to write an expression for the originally unvaporized LN_2 spray in terms of the effect of atomizing gas flow on $D_{v.5c}$.

Correlation of Dy, 5e with GN₂ Flowrate

1.5 cm axially downstream of the two fluid fuel nozzle. Values of the reciprocal volume median diameter, $D_{v.5e}^{-1}$ were measured and plotted against GN_2 flowrate, W_g , as shown in Fig. 4. Since the tests were conducted in high velocity gasflow, LN_2 jet breakup occurred primarily in the regime of aerodynamic stripping. From the plot shown in Fig. 5, the following correlation of reciprocal $D_{v.5e}$ with atomizing gas flowrate, W_g , was obtained:

The entire LN₂ spray cross section was sampled at

$$D_{v.5e}^{-1} = 301 W_g^{1.11}$$
 (1)

where $D_{v.5e}$ and W_g are expressed as cm⁻¹ and g/sec, respectively.

The exponent 1.11 for W_g is considerably less than 1.33, which is predicted by theory for liquid jet breakup in high velocity gasflow. This discrepancy is assumed to be due to a loss of small vaporizing LN_2 droplets, prior to spray measurements. In Ref. 2 it was found that at

a $\rm LN_2$ flowrate of 76.5 g/sec the exponent n agreed with atomization theory. However in the present study, the $\rm LN_2$ flowrate was set at a lower value, i.e., 51 g/sec. This was due to the fact that high $\rm LN_2$ flowrate readings often indicated excessive $\rm N_2$ vapor entrainment in the $\rm LN_2$ jets, which is detrimental to the atomization process because a high pressure drop and flowrate can cause "flash" boiling breakup of the liquid jet to occur at the nozzle orifice. This phenomena was not studied.

Experimental results obtained in the present study agree better with atomization theory than those reported in Ref. 2. This is due to the allowance made in the present study for the effect of droplet vaporization on volatile liquid dropsize measurements. This effect was not accounted for in Ref. 2. As a result, dropsize data did not give agreement with the theoretically predicted gasflow exponent, n, in the expression, $D_{v.5}^{-1} = k W_g^n$. However, the proportionality constant k was too low to adequately characterize the original spray. This was attributed to the fact that the study in Ref. 2 did not take into account the effect of small droplets vaporizing completely, before dropsize measurements could be taken.

LN₂ Droplet Velocity Calculations

To determine the effect of droplet vaporization rate on experimental values of $D_{v.5e}$, it was necessary to calculate vaporization time, t, by first calculating droplet velocity, V_d , for $D_{v.5e}$ over a distance of 0.13 cm in the nozzle throat and a distance of 2.2 cm from the nozzle orifice, x=0, to the downstream edge of the laser beam, as shown in Fig. 1. In order to determine volume-median drop velocity, V_d , the acceleration, a, of LN_2 droplets was calculated from the following momentum balance:

$$m_{d}a = \frac{1}{2}\rho_{g}A_{d}(V_{g} - V_{d})^{2}C_{d} \qquad (2)$$

where m_d and A_d are mass and area of dropsize $D_{v.5e}$, respectively, i.e., $m_d = \rho_1 \pi D_{v.5e}^3/6$, and C_d is drag coefficient for $D_{v.5e}$. Rewriting this expression, in terms of change in drop velocity squared, ΔV_d^2 , over the distance of travel, Δx , the following relationship is obtained;

$$\frac{\Delta V_d^2}{\Delta x} = \frac{3}{2} \frac{\rho_g}{\rho_1} \frac{\left(V_g - V_d\right)^2}{D_{v, f, h}} C_d \tag{3}$$

where $C_d = 27Re^{0.84}$, as given in Ref. 9, and Re is based on the characteristic dropsize, $D_{v.5}$.

To determine momentum transferred from GN_2 flow to the LN_2 droplets, the deceleration of an atomizing gaseous nitrogen jet into a surrounding low velocity airflow was determined as follows. At the nozzle orifice, GN_2 velocity, V_g was equal to the acoustic velocity, V_c , of gaseous nitrogen. Values of V_g that were used to solve Eq. (3), were calculated at downstream distances of x=0, 2.2, and 5 cm, respectively and plotted in Fig. 5. The calculated values of V_g^2 are based on data given in Ref. 13. These data are also plotted in Fig. 5 for comparison. The percent deceleration of the atomizing nitrogen gas is assumed to be approximately the same in both cases, since the two-fluid nozzles used in Ref. 13 and the present study were very similar in design.

By means of graphical integration, values of V_d^2 were calculated and plotted against downstream distance, x, as shown in Fig. 6. LN₂ droplet vaporization time Δt was calculated from this plot and the expression $\Delta t = x/V_d$. Calculated values of Δt for a given distance Δx are given in Table 2 along with calculated Reynolds numbers and drag coefficients averaged over the Δx distance. Gas and liquid transport properties used in calculating vaporization times are given in Table 3. Also, it should be noted that initially, in the nozzle throat, i.e., $\Delta x = -0.127$ to 0 cm, the reciprocal value of $D_{v.5}$ is the same as that given in Fig. 4 for $D_{v.5}$. Whereas at $\Delta x = 0$ to 2.2 cm, it decreased to the value of $D_{v.5}$.

Spray Vaporization Rate Calculations

The vaporization rate of cryogenic sprays characterized by $D_{v.5}$ were obtained by using the following heat-balance equation: $dm_d/dt = hA \Delta T/H_t$, where h is the heat-transfer coefficient, A is droplet surface-area, based on $D_{v.5}$, $\Delta T = T_g - T_d$, and $H_t = Hv + Cp \Delta T$, where H_v is the latent heat of vaporization of liquid nitrogen and C_p is the specific heat of gaseous nitrogen. This expression may be rewritten as follows to obtain vaporization rate in terms of changes in droplet surface-area with time:

$$dD_{v,5}^{2}/dt = 4k_{\sigma} \Delta T Nu/\rho_{1}H_{t}$$
 (4)

where k_g and ρ_1 are gas thermal conductivity and liquid density, respectively. In a previous fuel-droplet study using a high-speed droplet tracking camera to determine vaporization rates for fuels such as n-octane and methanol, as well as numerous other liquids includ-

ing water, benzene, acetone, nitrobenzene, butanol, and carbon tetrachloride, it was found that: Nu = 2 + 0.303 Re^{0.6} where Re = ρ_g D_{v.5} $\Delta V/\mu_g$, and ΔV is the average velocity difference over an incremental distance Δx .

To calculate vaporization time, t, inside the twofluid nozzle, an average value of V_d^2 was taken between x = -0.13 and 0 cm, and used to calculate Δt , i.e., $\Delta t = \Delta x / \overline{V}_d$. In a similar manner, the average Reynolds number, Re, was determined by averaging Reynolds numbers over a distance of 0.13 cm. Re and t were then used in heat-transfer calculations to determine LN2 droplet vaporization rates. Calculations were based on characteristic diameters $D_{v.5c}$, $D_{v.5e}$ and transport properties listed in Table 3. GN₂ viscosity and thermal conductivity were evaluated at the average gas-film temperature, i.e., $T_i = 1/2(T_g + T_1)$. Droplet surface temperatures were assumed to be near the boiling point, 77 K, while droplet sprays were being accelerated and partially vaporized. The latent heat of vaporization of LN₂ was evaluated at 77 K and the specific heat of nitrogen vapor was evaluated at the average gas-film temperature, T_f.

Calculated values of the change in characteristic droplet diameter squared, $D_{v,5}^2$, that occurs due to partial vaporization of the sprays, are given in Table 4. The calculation was made for gaseous-nitrogen and liquid-nitrogen flowrates of 1.82 and 51 g/sec, respectively. Calculated values of $D_{v.5c}^2$, were obtained from the expression; $-\Delta D_{v.5}^2 = D_{v.5c}^2 - D_{v.5e}^2$. Values of $D_{v.5e}^{-1}$ are plotted against GN_2 flowrates as shown in Fig. 4. From this plot, the following correlating expression is obtained for calculated values of $D_{v.5c}$;

$$D_{v,5c}^{-1} = 285 W_g^{1.33}$$
 (5)

In terms of the Sauter mean diameter, D_{32} , it was found that Eq. (5) can be rewritten as; $D_{32c}^{-1} = 422 \text{ W}_g^{1.33}$.

Comparison of Eq. (5) with Eq. (1), shows that the exponent 1.33 for gas flowrate in Eq. (5) is considerably greater than 1.11, as given in Eq. (1) for experimental values of $D_{v.5e}$. Equation (5) is in good agreement with atomization theory and the proportionality constant, 285 is only slightly less than the value of 301 given in Eq. (1). Thus, droplet vaporization had considerable effect on the exponent, n, but only a small effect on the constant, k. Also, it should be noted that Eq. (5) is useful in establishing the initial values of $D_{v.5}$ that are needed in analytical modeling of cryogenic

liquid spray vaporization and combustion, on a macroscopic scale.

The expression, $D_{v.5}^{-1}=k_c$ W_g^n used in the present study of LN_2 sprays cannot be directly compared with expressions obtained in previous studies of water sprays, until the effect of atomizing-gas temperature on LN_2 spray formation has been investigated and determined. In the case of LN_2 and water sprays, the exponent n should equal 1.33 in agreement with atomization theory. However, the value of k will be considerably different due to large differences in their properties. The effect of atomizing-gas properties on LN_2 jet breakup also needs to be further investigated.

Concluding Remarks

It was possible to determine the effect of droplet vaporization on cryogenic spray measurements because previous studies in Ref. 9 were able to provide heat and momentum transfer expressions for droplets accelerating and vaporizing in high velocity gasflows. As a result, an originally unvaporized volume-mean diameter, $D_{v.5c}$, could be calculated for LN_2 sprays produced with two-fluid fuel nozzles. The results agree well with atomization theory and can be readily applied to analytical modeling of fuel spray combustion in gas turbine and rocket combustors, under the conditions used in the present study. Dropsize data for different atomizing gases and wide ranges of gas temperature are needed in order to generalize the expressions derived in this study of GN_2 and LN_2 .

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TABLE 1.—ATOMIZING-GAS VELOC-ITY EXPONENT, n, FOR LIQUID-NITROGEN JET BREAKUP,

 $D_c \sim V_{\pi}^{-n}$

Exponent
1.33
1.00
1.14
1.33
^a 1.33
1.33
1.33

^aWax sphere dropsize data.

TABLE 2.—CALCULATED LN₂ DROPLET VAPORIZATION TIME, ΔT , FOR $D_{v.5}$,

AT $W_{\pi} = 1.82 \text{ g/sec}$

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Zone	Δx, cm	Re ^a	$\mathbf{C_d}$	Δtx10 ⁴ , sec
Inside nozzle throat	0.13	70	0.76	0.47
Downstream of nozzle	2.2	35	1.36	3.26
Total	2.33			3.73

$$[\]label{eq:Re_alpha} \begin{split} ^{\mathbf{a}}\overline{\mathbf{R}}\mathbf{e} &= \mathbf{R}\mathbf{E_1} \ \Delta\mathbf{t_1}/\mathbf{t_t} + \mathbf{R}\mathbf{e_2} \ \Delta\mathbf{t_2}/\mathbf{t_t}. \\ ^{\mathbf{b}}\mathbf{Inside} \ \mathbf{fuel} \ \mathbf{nozzle}. \end{split}$$

TABLE 3.—ATOMIZING-GAS AND LIQUID-JET TRANSPORT PROP-

ERTIES, AT $W_g = 4.54 \text{ g/sec}$,

$$T_g = 293 \text{ K}, T_1 = 77 \text{ K},$$

AND $x = -0.127 \text{ cm}$

Nitrogen gas	
V cm/sec	3.43×10^4
ρ _g , cm/sec	3.84×10^4
μ_{g} , g/cm sec	1.25×10^4
a μ _g , g/cm sec a k _g , cal/sec sq cm (°C/cm)	4.20×10^{5}
LN ₂ droplets	
V _d , cm/sec	204
ρ_1 , g/cc	0.80
H, cal/g	47.8
C _{pv} , cal/g °C	0.25

^aEvaluated at $T_f = 1/2(T_g - T_1)$.

TABLE 4.—REDUCTION IN DROPSIZE DUE TO VAPORIZATION, $-\Delta D_{y,5}^2$ AND CALCULATED ORIGINAL

DROPSIZE, D_{v.5}

	W _g , g/sec	$-\Delta D_{v.5_2}^2 \times 10^8,$	$D^2_{v.5e} \times 10^8,$ cm ²	D-1 cm-f'
	1.82	37.0	289	630
	2.72	21.2	113	1044
I	3.63	19.3	62	1530
	4.54	17.6	39	2162

$$^{a}D_{v,5c}^{-1} = (D_{v,5e}^{2} - \Delta D_{v,5c}^{2})^{-0.5}.$$

^cOutside of fuel nozzle.

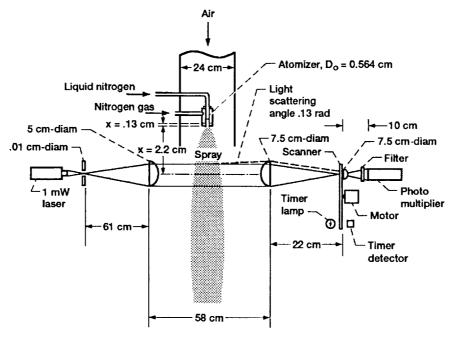


Figure 1.—Atmospheric pressure test section and optical path of scattered-light scanner.

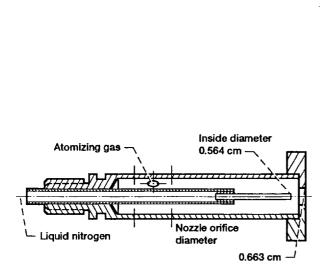


Figure 2.—Diagram of pneumatic two-fluid atomizer.

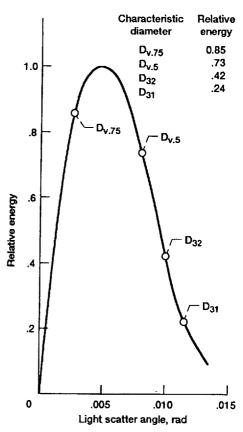


Figure 3.—Typical scattered-light energy curve with four characteristic diameters.

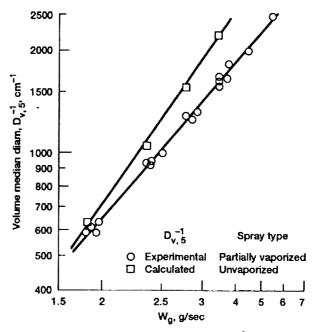


Figure 4.—Effect of atomizing-gas flowrate on $D_{\nu,\ 5}^{-1}$, for partially vaporized and unvaporized cryogenic-liquid sprays.

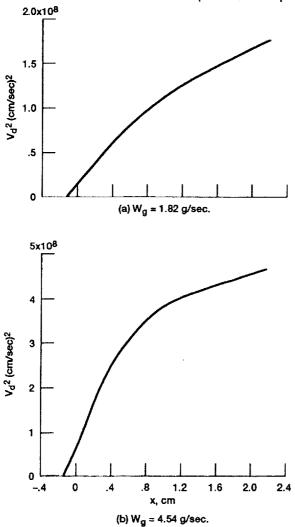


Figure 5.—Acceleration of characteristic LN₂ dropsize, D_{v, 5}.

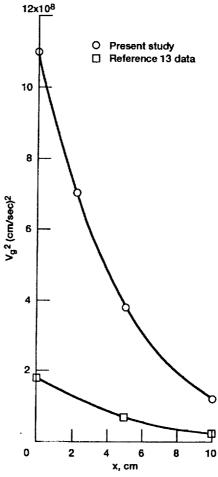


Figure 6.—Deceleration of atomizing gas downstream of fuel nozzle.

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gasflow, was experimentally dropsize, $D_{v.5}$, of LN_2 sprays Under sonic gas-velocity confollowing correlation of voluatomizers; with atomizing a constant k_c and exponent navestigated and it was found tion theory. This was attributed the obtained experimentally: D_v Values of $D_{v.5}$, that existed property was derived for originally under the sonic property of	Iti-phase flow breakup of liquid investigated. A scattered-light s and to determine the effect of denditions, liquid-jet breakup occur ime-median drop diameter, $D_{v,5}$, gas flowrate, W_g , was derived for are functions of droplet vaporization that $n = 1.11$, which is considerated to the evaporative loss of ver- $\frac{1}{v,5} = 301 W_g^{-1.11}$. Values of $D_{v,5}$, which is considerated to the evaporative loss of the partial vaporization of the	scanning instrument was a roplet vaporization on experred in the regime of aeroc with atomizing gas flowrater two-fluid atomizers; Detion rate. In this study, put bly less than the value of a ry small droplets. As a result, that existed prior to particle LN ₂ sprays, were calculated $M_{\pi}^{1.33}$. This express	cted into sonic velocity nitrogen used to measure the characteristic erimental dropsize measurements. dynamic stripping. As a result, the ate, W_g , was derived for two-fluid $_{v,5}^{-1} = k_c W_g^n$, where proportionally artially vaporized sprays were in-1.33 that is predicted by atomizasult, the following expression was all vaporization of the LN ₂ sprays: ated and the following expression sion agrees well with atomization		
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